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(73) Proprietor: **XEROX CORPORATION**
Xerox Square - 020
Rochester New York 14644(US)

(72) Inventor: **Weller, Ivor**
5316 Camino Alta Mira
Castro Valley California 94546(US)
Inventor: **Waibel, Helmut K.**
227 Yerba Buena Place
Fremont California 94536(US)

(74) Representative: **Goode, Ian Roy et al**
Rank Xerox Patent Department
Albion House
55 New Oxford Street
London WC1A 1BS (GB)

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Description

This invention relates to a serial impact printer and, more particularly, to a platen support arrangement capable of withstanding rapidly repeating, long duration, high impact loads without generating extraneous noise.

The office has, for many years, been a stressful environment due, in part, to the large number of objectionable noise generators, such as typewriters, high speed impact printers, paper shredders, and other office machinery. Where several such devices are placed together in a single room, the cumulative noise pollution may even be hazardous to the health and well being of its occupants. The situation is well recognized and has been addressed by governmental bodies who have set standards for maximum acceptable noise levels in office environments. Attempts have been made by the technical community to reduce the noise pollution. Some of these methods include enclosing impact printers in sound attenuating covers, designing impact printers in which the impact noise is reduced, and designing quieter printers based on non-impact technologies such as ink jet and thermal transfer.

Noise measurements are often referenced as dBA values. The "A" scale, by which the sound values have been identified, represents humanly perceived levels of loudness as opposed to absolute values of sound intensity. When considering sound energy represented in dB (or dBA) units, it should be borne in mind that the scale is logarithmic and that a 10 dB difference means a factor of 10, a 20 dB difference means a factor of 100, 30 dB a factor of 1000, and so on.

Typically, impact printers generate impact noise in the range of 65 to just over 80 dBA, which is deemed to be intrusive. When reduced to the high 50s dBA, the noise is construed to be objectionable or annoying. It would be highly desirable to reduce the impact noise to a dBA value in the vicinity of 50 dBA. For example, the IBM Selectric ball unit typewriters generate about 78 dBA, while the Xerox Memorywriter typewriters generate about 68 dBA. The typewriter of the present invention has been typically measured at slightly less than 52 dBA. This represents a dramatic improvement on the order of about 100 times less noisy than present day offices, a notable achievement toward a less stressful office environment.

Although the printing impact, produced as the hammer impacts and drives the type character pad against the ribbon, the print sheet and the platen with sufficient force to release the ink from the ribbon, is the major source of noise in the typewriter, other noise sources are present. In the presently available typewriters, the impact noise over-

shadows the other noises. But, once the impact noise has been substantially reduced, the other noises will no longer be extraneous. Thus, the design of a truly quiet printer requires the designer to address reducing all other noise sources, such as those arising from platen vibration, carriage motion, character selection, ribbon lift and advance, as well as from miscellaneous clutches, solenoids, motors and switches.

In conventional ballistic hammer impact printers a hammer mass of about 2.5 grams is driven ballistically by a solenoid-actuated clapper toward the ribbon/paper/platen combination. When the hammer hits the rear surface of the character pad it drives it against the ribbon/paper/platen combination and deforms the platen surface which, when it has absorbed the hammer impact energy, seeks to return to its normal shape by driving the hammer back to its home position where it must be stopped, usually by another impact. This series of impacts is the main source of the objectionable impact noise.

Looking solely at the platen deformation impact portion of the hammer movement, the total dwell time is typically in the vicinity of 100 microseconds. At a printing speed of 30 characters per second, the mean time available between character impacts is about 30 milliseconds. The impact noise reduction achieved by the printing mechanism of the present typewriter is made possible by significantly stretching the impact dwell time to a substantially larger fraction of the printing cycle than is typical in conventional printers. For instance, if the dwell time were stretched from 100 microseconds to 6 to 10 milliseconds, this would represent a sixty- to one hundred-fold increase, or stretch, in pulse width relative to the conventional. By extending the deforming of the platen over a longer period of time, the resonant frequency is proportionally decreased and an attendant reduction in noise output can be achieved. In a resonant system, since the mass is inversely proportional to the square of the frequency shift, a massive increase in hammer mass is required.

The general concept implemented in the present typewriter, i.e. reduction of impulse noise achieved by stretching the deformation pulse, has been recognized for many decades. As long ago as 1918, in U.S. Patent No. 1,261,751 (Anderson) it was recognized that quieter operation of the printing function in a typewriter may be achieved by increasing the "time actually used in making the impression". A type bar typewriter operating upon the principles described in this patent was commercially available at that time.

The quiet impact printing mechanism incorporating the present invention is described, and its theory of operation is explained in the following

commonly assigned patents. US-A-4,668,112 (Gabor *et al*), entitled "Quiet Impact Printer", relates to the manner in which the impact force in a printer of this type is controlled; US-A-4,673,305 (Crystal), entitled "Printwheel For Use in a Serial Printer", relates to a printwheel modified for quiet operation when used with an alignment member; US-A-4,678,355 (Gabor *et al*) entitled "Print Tip Contact Sensor for Quiet Compact Printer", relates to an impacting element having a sensor thereon for signaling initiation of impact; US-A-4,681,469 (Gabor), entitled "Quiet Impact Printer", relates to the high mass, prolonged contact period, parameters of a printer of this type; US-A-4,686,900 (Crystal *et al*), entitled "Impact Printer With Application of Oblique Print Force", relates to a shear inducing impacting element; and US-A-4,737,043 (Gabor *et al*), entitled "Impact Mechanism for Quiet Impact Printer", relates to a unique prime mover and high mass print tip driver.

It will become apparent from a review of the above-identified, patents that a character impacting member, having a high effective mass, is driven with a first force, from a starting position to the rear of a character element and then continues forward, together with the character imprinting element, which picks up the marking (or correcting) ribbon, across a throat gap into incipient contact with the platen/paper combination. A second force, of a magnitude sufficient to release the marking material and deform the platen, is applied as soon as the control microprocessor receives a signal that contact has been made with the platen. The second force, having a magnitude of about 176.58N (18 Kg) and a duration of about 10 milliseconds, would overload conventional platen mounting arrangements which usually comprise seats formed in the sheet metal side walls, probably causing deformation of the side walls and rendering the printer useless. Furthermore, the conventional seats would not restrain the substantial rebound of the platen which would vibrate and generate noise.

Therefore, it is the primary object of the present invention to provide a platen support implementation which will hold the platen in place in seats capable of sustaining the massive, long duration, imprinting force and retain its dimensional integrity and which will retain the platen shaft within its supporting seats during platen flexures, so as to prevent noise caused by unrestrained vibration.

It is a further object of this invention to provide a platen support arrangement which prevents lateral platen vibration and lateral platen creep.

Accordingly, the present invention provides a serial impact printer comprising a platen for supporting an image receptor, said platen including a central support shaft and a cylindrical impact absorbing sleeve secured thereto, a print element

having character imprinting portions disposed thereon, a print element selector for moving said print element to position a selected character portion at a printing position, a marking ribbon positionable between said print element and said platen, and means for delivering a force to said character portion so as to drive it against said platen, said force delivering means having an effective mass of at least 0.23 Kg and being arranged so as to deform said platen for a contact period of at least 1 millisecond, and wherein said printer is characterized by means for capturing the ends of said shaft, said means including support means defining seats for receiving the ends of the shaft, said support means being capable of withstanding, without distortion, an impact printing force from said force delivering means, delivered in a first direction, and resilient means for retaining the ends of said shaft in contact with the seats with a force delivered in the first direction sufficient to overcome rebound movement of the ends of said shaft in a second direction substantially opposite to said first direction.

Other objects and further features and advantages of this invention will be apparent from the following, more particular description considered together with the accompanying drawings, wherein:

Figure 1 is a partial perspective view schematically showing the relevant features of a quiet impact printer in which the present invention may be incorporated;

Figure 2 is a partial elevation view showing the platen mounting arrangement;

Figure 3 is a sectional view taken substantially along line 3-3 of Figure 2;

Figure 3a is an enlarged schematic view showing the platen support; and

Figure 4 is a schematic plan view of the platen showing the effect of imprinting forces thereon.

Salient features of the quiet impact printer, in which the present invention is incorporated, are shown in Figure 1. These include a platen 10 suitably mounted on a frame (not shown) for rotation about its axis 11 to advance and retract a record carrier, comprising a single sheet or a multi-part form, on which characters may be imprinted. A carriage support beam 12, integral with a body casting (not shown), fitted with rod stock rails 14, spans the printer from side-to-side beneath and parallel to the platen for rigidly and smoothly supporting a carriage for traversing movement parallel to the axis of the platen. Secured upon the carriage, for traversing movement therewith, is a horseshoe-shaped interposer 16 mounted for arcuate movement about pivot axis 18 and carrying a print tip 20 at its apex.

A rockable bail bar 22 extends substantially parallel to the axis of platen 10 and is constrained

to limited angular movement toward and away from the platen about its axis of rotation 24. Prime mover 26 in the form of a reciprocating voice coil motor, a rotary motor, or any other suitable driver, is connected to the bail bar for imparting the rocking movement thereto. A bead or rail 28 on the bail bar receives one end of push rod 30, via a pair of capturing rollers 32 and 34, for moving it toward and away from the platen as the bail bar is rocked. The non-collapsible push rod, as illustrated, is a generic form of this element which is collapsible in its commercial form for allowing the print tip 20 to be drawn back away from the platen in order to allow the operator easy access to a printwheel. A bearing surface (not shown) on the opposite end of the push rod is biased into engagement with a seat on the rear wall of the print tip 20 by means of tension springs 38 extending between pins 40 on the push rod and suitable anchors on the interposer 16. The drive force of the prime mover 26 is multiplied by the bail bar 22 and is translated to the print tip 20 by the push rod 30 which may pivot about its bearing surface 36, so as to enable it to follow the arcuate path of the print tip, prescribed by the pivoting interposer 16. Thus, the print tip delivers the approximately 176.58N (18 Kg) impact force, having a duration of about 10 milliseconds, to the platen at an oblique angle in the range of about 15° to 40°.

Pivot frame 42 is also mounted upon the carriage for pivoting movement toward and away from the platen in an arcuate path about pivot axis 44. A printwheel drive motor 46, having a drive shaft terminating in a coupling member (not shown), is secured to the pivot frame for movement therewith. When the operator drops a printwheel 48 into the printer, it is captured between a retaining member and the coupling member and moves together with the pivot frame.

Turning to Figures 2 and 3, the platen mounting arrangement of the present invention is shown in detail. Platen 10 extends substantially completely between printer side walls 50 and 52, although it is not supported thereon, as is done in conventional printers. The platen utilized in our quiet printer is more rigid than conventional platens in order that it may withstand the high impact forces delivered by the print tip 20. It comprises a central tubular steel body 54 surrounded by a resilient sleeve 56, end caps 58 secured in the body, a platen shaft 60 passing through and secured in the end caps, and bearings 62 and 64 mounted upon the shaft adjacent the end caps and held in place by retainer washers 66 and 68. Bearing 62 is formed with flanges 69. One end 70 of the platen shaft is bullet-shaped and the other end supports a drive coupling gear 72 whose hub 74 is secured to the shaft. The bullet-shaped platen end 70 is axially

biased away from side wall 52 by button 76 supported upon side wall mounted stud 78 and biased by compression spring 80.

A body casting, which includes the carriage support beam 12, underlying the platen, has a pair of rigid upright posts 82, one on either end of the platen, each formed with a wedge-shaped saddle 84 for receiving bearings 62 and 64. Each of the platen bearings is urged against its respective saddle by a biasing mechanism as illustrated in Figures 3 and 3a (portions thereof are also shown in Figure 2). Each biasing mechanism includes a platen shaft basing roller 86 pinned for rotation at the upper end of a generally U-shaped sheet metal pivoting plate 88, which in turn is mounted for rotation about stud 90 secured to side wall 52. Heavy-duty tension spring 92 is anchored to the side wall at one end and is secured to the pivoting plate at its other end, so as to urge the biasing roller against the platen shaft 60. At the lower end of the pivoting plate a there is mounted a cam follower stud 94 acted upon by the camming surface 96 of pivotable release lever 98, mounted for rotation about pivot pin 100.

In order to release the platen for removal, the operator or repairman must overcome the heavy basing force of spring 92. By applying a downward spring release force, in the direction of arrow A, against release lever actuating surface 102, the release lever 98 is rotated about pivot pin 100. Camming surface 96 raises cam follower stud 94, to rotate pivoting plate 88 in the direction of arrow B against the force of the tension spring 92. Rotation of the release lever through an angle of about 45° brings the cam seat 104 into position to receive and hold the cam follower stud 94 in an over-center position so that the pivoting plate is arrested in its platen shaft release position. Raising the actuating surface 102 frees the cam follower stud from the cam seat and allows the tension spring 92 to once again urge the pivoting plate in a counterclockwise direction about stud 90, driving the platen shaft basing roller 86 against the platen shaft 60.

This heavy duty platen mounting arrangement is necessary to withstand the high impact forces applied over the relatively long duration. Both static and dynamic conditions have been taken into consideration in this unique design. The static loading condition is easiest to visualize. A high force of about 176.58N (18 Kg) pushing against the platen must be resisted with a structure capable of withstanding it. To this end we have provided the cast metal standards 82 having saddles formed therein.

To understand the dynamic loading condition, the illustration of Figure 4 is referenced. A series of rapidly applied, long duration, high impact forces is delivered to the platen all along its length during

serial printing. Consideration will be given to the most extreme condition, wherein the impact forces are rapidly applied in the vicinity of the center of the platen. In this case, when viewed as a free body in space, the platen will be seen to be flexed by the force F at its center and will pivot about (be stationary at) nodes N_A and N_B located about $\frac{1}{4}$ of the platen length on either side of the center. Under this condition, it can be seen that the ends of the platen will thrust forward with forces f_A and f_B directed oppositely to force F (-F direction). As the impacting element is drawn back by the prime mover, the platen follows it and deforms in the opposite direction until contact is released and then the platen will rebound back in the direction of initial force F. As its vibrations are damped, the platen ends have a tendency to move out of the axial plane toward the front of the printer each time the platen flexes in the direction of force F.

It should be apparent that in order to maintain the platen in its axial plane, rigidity is required both in the F and -F directions, so as to prevent the platen from bouncing in its seats and generating noise. We have accomplished this, in the F direction, with the saddles 84 in the standards 82 of the body casting, and in the -F direction with the outboard platen shaft biasing rollers 86 which push on the shaft with a spring force of about 52.97N (5.4 Kg) (indicated by arrow Z in Figure 3a). A spring force of a magnitude selected to be adequate for holding the platen shaft ends in the saddles when a force is being applied will be sufficient to hold the ends in place during the rebound flexure.

Another source of noise is the rattle generated between the platen shaft and its bearings 62 and 64. We eliminate that noise, by removing all play between the bearings and the shaft as shown by the three-point support in Figure 3a. By pushing the biasing rollers 86 against the shaft 60, adjacent to and outboard of the saddles 84, we exert a force on the inner diameter of the bearings, which in turn push the outer diameter of the bearing into the saddles. This mounting arrangement also insures that the platen seats itself correctly and will always be accurately positioned.

When the platen is placed in the printer, the bullet end 70 of shaft 60 snaps into a recess (not shown) in the apex of spring biased button 76 for positioning the platen in the correct axial position. As the button pushes the platen shaft in the direction of arrow C it urges the left end cap 58 against the outer surface of flange of bearing 62 surface of the flange against the side wall of post 82 for eliminating noise generating axial play between the bearings and the platen shaft. The spring biased button 76 provides a positive load to the platen in one direction for preventing axial migration thereof,

which undermines accurate correction. We have provided the flange 69 on bearing 62 to act as a stop to limit axial movement. Absent the bias load, as the platen is rotated there is a possibility that it will migrate axially under the influence of the cradle rollers beneath the platen (not shown) on which it is seated. If one of the cradle rollers is slightly cocked it would have a tendency to drive the platen in one axial direction when the platen is rotated in one direction and in the opposite axial direction when the platen is oppositely rotated. When the correction command is given, it is likely that the platen will have to back up one or more lines and that there will be a slight displacement between the axial positioning of the printed character and the correcting character. Any such displacement will prevent lift-off of the entire character. Thus, in addition to the noise reducing benefits of the axial loading device, it will also introduce a uniform axial displacement regardless of the direction of platen rotation.

It should be understood that the present disclosure has been made only by way of example and that numerous changes in details of construction and the combination and arrangement of parts may be resorted to without departing from the scope of the invention as hereinafter claimed.

Claims

1. A serial impact printer comprising a platen 10 for supporting an image receptor, said platen including a central support shaft 60 and a cylindrical impact absorbing sleeve 54, 56 secured thereto, a print element 48 having character imprinting portions disposed thereon, a print element selector 46 for moving said print element to position a selected character portion at a printing position, a marking ribbon positionable between said print element and said platen, and means 30 for delivering a force to said character portion so as to drive it against said platen, said force delivering means having an effective mass of at least 0.23 Kg and being arranged so as to deform said platen for a contact period of at least 1 millisecond, and wherein said printer is characterized by

means for capturing the ends of said shaft 60, said means including support means 82 defining seats 84 for receiving the ends of the shaft 60, said support means 82 being capable of withstanding, without distortion, an impact printing force from said force delivering means, delivered in a first direction, and resilient means 86, 88 for retaining the ends of said shaft 60 in contact with the seats 84 with a force delivered in the first direction sufficient to

- overcome rebound movement of the ends of said shaft in a second direction substantially opposite to said first direction.
2. The serial impact printer as defined in claim 1 wherein said support means comprises a support block 82 extending from the base of said printer at each end of said shaft, each block defining a wedge-shaped seat 84 therein. 5
 3. The serial impact printer as defined in claim 2 wherein said ends of said shaft 60 support bearings 62, 64 thereon and said bearings are held against said wedge-shaped seats 84 by said resilient means 86, 88. 10
 4. The serial impact printer as defined in any one of claims 1 to 3 wherein said resilient means comprises a spring biased roller 86 at each end of said shaft 60, each of said rollers being movable from a first position for capturing an end of said shaft to a second position for releasing said end of said shaft. 15
 5. The serial impact printer as defined in claim 4 wherein each of said rollers 86 is urged directly against said ends of said shaft 60 so as to remove any clearance from between said shaft 60 and said bearings 62, 64. 20
 6. The serial impact printer as defined in claim 5 wherein each of said rollers is urged directly against said ends of said shaft, outboard of said bearings 62, 64 supported on the ends of said shaft for seating in said wedge-shaped seats, and 25
 7. The serial impact printer as defined in any one of claims 4 to 6 wherein said resilient means comprises a pivotable arm 88 on which said roller 86 is mounted, a spring 92 connected to said arm for urging said arm in the capturing direction, and a latch arm 98 for moving said pivotable arm in the release direction, opposite to said capturing direction. 30
 8. The serial impact printer as defined in claim 7 wherein said latch arm 98 is pivotable and carries a driving cam 96 thereon for interacting with a cam follower 94 on said pivotable arm 88 for moving said pivotable arm in the release direction. 35
 9. A serial impact printer comprising a platen 10 for supporting an image receptor, said platen including a central support shaft 60 and a cylindrical impact absorbing sleeve 54, 56 secured thereto, a print element 48 having character imprinting portions disposed thereon, a print element selector 46 for moving said print

element to position a selected character portion at a printing position, a marking ribbon positionable between said print element and said platen, and means 30 for delivering a force to said character portion so as to drive it against said platen, said force delivering means having an effective mass of at least 0.23 Kg and being arranged so as to deform said platen for a contact period of at least 1 millisecond, and wherein said printer is characterized by

means for capturing the ends of said shaft 60, said means for capturing including support blocks 82 extending from the base of said printer at each end of said shaft, each block defining a wedge-shaped seat 84 therein, said blocks being capable of withstanding, without distortion, an impact printing force from said force delivering means, delivered in a first direction, and means 86, 88 for retaining the ends of said shaft 60 in contact with said support blocks 82 with a force sufficient to overcome movement of the ends of said shaft in a second direction substantially opposite to said first direction,

bearings 62, 64 supported on the ends of said shaft for seating in said wedge-shaped seats, and

means 76, 80 for biasing said shaft axially so as to eliminate axial play between said shaft and said bearings and so as to eliminate axial platen creep by providing a driving force to urge said shaft in one axial direction.

Patentansprüche

1. Zeichenschlagdrucker mit einem Schreibträger (10) zum Halten eines Bildaufnahmeelementes, wobei der Schreibträger eine zentrale Haltewelle (60) und eine zylindrische, schlagabsorbierende Hülse (54, 56) hat, die daran befestigt ist, einem Druckelement (48) mit daran angeordneten Zeichenindruckabschnitten, einer Druckelementwähleinrichtung (46) zum Bewegen des Druckelements, um einen ausgewählten Zeichenabschnitt an einer Druckposition anzugeordnen, einem Farbband, das zwischen dem Druckelement und dem Schreibträger angeordnet werden kann, und einer Einrichtung (30) zur Abgabe einer Kraft auf den Zeichenabschnitt, um diesen gegen den Schreibträger zu bewegen, wobei die Kraftabgabeeinrichtung eine wirksame Masse von wenigstens 0,23 kg hat und so angeordnet ist, daß sie den Schreibträger über eine Kontaktzeitspanne von wenigstens 1 ms verformt, und wobei der Drucker gekennzeichnet ist durch eine Einrichtung zum Ergreifen der Enden der

- Welle (60), wobei die Einrichtung eine Halteeinrichtung (82) bildende Sitze (84) zur Aufnahme der Enden der Welle (60) enthält und die Halteeinrichtung (82) in der Lage ist, ohne Formänderung einer Schlagdruckkraft von der Kraftabgabeeinrichtung zu widerstehen, die in einer ersten Richtung abgegeben wird, und durch elastische Mittel (86, 88) zum Halten der Enden der Welle (60) in Kontakt mit den Sitzen (84) mit einer Kraft, die in der ersten Richtung abgegeben wird und ausreicht, die Rückprallbewegung der Enden der Welle in der zweiten Richtung zu überwinden, die im wesentlichen der ersten Richtung entgegengesetzt ist.

Zeichenschlagdrucker nach Anspruch 1, wobei die Halteeinrichtung einen Halteblock (82) aufweist, der sich von der Basis des Druckers an jedem Ende der Welle erstreckt, wobei jeder Block einen keilförmigen Sitz (84) darin bildet.

Zeichenschlagdrucker nach Anspruch 2, wobei die Enden der Welle (60) daran Lager (62, 64) halten, und wobei die Lager gegen die keilförmigen Sitze (84) mittels elastischer Einrichtungen (86, 88) gehalten sind.

Zeichenschlagdrucker nach jedem der Ansprüche 1 bis 3, wobei die elastischen Einrichtungen eine federgespannte Rolle (86) an jedem Ende der Welle (60) aufweisen und jede der Rollen aus einer ersten Position zum Erfassen eines Endes der Welle in eine zweite Position bewegbar ist, um das Ende der Welle freizugeben.

Zeichenschlagdrucker nach Anspruch 4, wobei jede der Rollen (86) direkt gegen die Enden der Welle (60) gezwängt wird, um jeglichen Abstand zwischen der Welle (60) und den Lagen (62, 64) zu beseitigen.

Zeichenschlagdrucker nach Anspruch 5, wobei jede der Rollen (86) direkt gegen die Enden der Welle (60) außerhalb der Lager (62, 64) gezwängt wird.

Zeichenschlagdrucker nach jedem der Ansprüche 4 bis 6, wobei die elastische Einrichtung einen schwenkbaren Arm (88) aufweist, an der die Rolle (86) befestigt ist, eine Feder (92), die mit dem Arm verbunden ist, um den Arm in der Erfassungseinrichtung zu zwängen, und einen Verriegelungsarm (98), um den schwenkbaren Arm in die Freigaberichtung, entgegengesetzt zu der Erfassungsrichtung, zu bewegen.

Zeichenschlagdrucker nach Anspruch 7, wobei der Verriegelungsarm (98) schwenkbar ist und einen Antriebsnocken (96) daran trägt, der mit einem Nockenfolger (94) an dem schwenkbaren Arm (88) zum Bewegen des schwenkbaren Arms in die Freigaberichtung zusammenwirken kann.

Zeichenschlagdrucker mit einem Schreibträger (10) zum Halten eines Bildaufnahmeelementes, wobei der Zeichenträger eine zentrale Haltewelle (60) und eine zylindrische, schlagabsorbierende Hülse (54, 56) aufweist, die daran befestigt ist, einem Druckelement (48), an dem Zeicheneindruckabschnitte angeordnet sind, einer Druckelementauswahlleinrichtung (46) zum Bewegen des Druckelementes, um einen ausgewählten Zeichenabschnitt an einer Druckposition anzurordnen, einem Farbband, das zwischen dem Zeichenelement und dem Schreibträger angeordnet werden kann, und mit einer Einrichtung (30) zur Abgabe einer Kraft auf den Zeichenabschnitt, um diesen gegen den Schreibträger zu bewegen, wobei die Kraftabgabeeinrichtung eine wirksame Masse von wenigstens 0,23 kg hat und so angeordnet ist, daß sie den Schreibträger über eine Kontaktzeitspanne von wenigstens 1 ms deformiert, wobei der Drucker gekennzeichnet ist durch eine Einrichtung zum Einfangen der Enden der Welle (60), wobei die Einrichtung zum Einfangen Halteblocks (82) enthält, die sich von der Basis des Druckers an jedem Ende der Welle erstrecken und wobei jeder Block darin einen keilförmigen Sitz (84) bildet, wobei die Blocks in der Lage sind, ohne Verformung einer Schlagdruckkraft von der Kraftabgabeeinrichtung standzuhalten, die in einer ersten Richtung abgegeben wird, und Einrichtungen (86, 88) zum Halten der Enden der Welle (60) in Kontakt mit den Halteblocks (82) mit einer ausreichenden Kraft, um eine Bewegung der Enden der Welle in einer zweiten Richtung zu überwinden, die im wesentlichen der ersten Richtung entgegengesetzt ist, Lager (62, 64), die an den Enden der Welle gehalten sind, um in den keilförmigen Sitzen aufzuliegen, und eine Einrichtung (76, 80) zum axialen Vorspannen der Welle, um ein axiales Spiel zwischen der Welle und den Lagen zu beseitigen und um ein axiales Verlagern des Schreibträgers durch Bereitstellen einer Antriebskraft zu vermeiden, die die Welle in einer axialen Richtung vorspannt.

Revendications

1. Imprimante à impact sérielle comprenant une platine (10) pour supporter un récepteur d'image, ladite platine comportant un arbre central de support (60) et un manchon cylindrique absorbant les impacts (54, 56) qui lui est fixé, un élément d'impression (48) sur lequel sont disposées des parties d'impression de caractères, un sélecteur d'élément d'impression (46) pour déplacer ledit élément d'impression de manière à placer une partie sélectionnée d'un caractère à une position d'impression, un ruban de marquage pouvant être placé entre ledit élément d'impression et ladite platine, et un moyen (30) pour fournir une force à ladite partie de caractère de manière à l'entraîner contre ladite platine, ledit moyen de fourniture de force ayant une masse effective d'au moins 0,23 kg et étant agencé de manière à déformer ladite platine pendant une période de contact d'au moins 1 milliseconde, et où ladite imprimante est caractérisée par :

un moyen pour capturer les extrémités dudit arbre (60), ledit moyen comportant un moyen de support (82) qui définit des sièges (84) pour recevoir les extrémités de l'arbre (60), ledit moyen de support (82) étant capable de résister, sans déformation, à une force d'impression par impact en provenance dudit moyen de fourniture de force, délivrée dans une première direction, et un moyen élastique (86, 88) pour maintenir les extrémités dudit arbre (60) en contact avec les sièges (84) avec une force fournie dans la première direction qui est suffisante pour surmonter le mouvement de rebondissement des extrémités dudit arbre dans une seconde direction sensiblement opposée à ladite première direction.
2. Imprimante à impact sérielle selon la revendication 1, dans laquelle ledit moyen de support comprend un bloc de support (82) s'étendant à partir de la base de ladite imprimante à chaque extrémité dudit arbre, chaque bloc définissant intérieurement un siège (84) en forme de cale.
3. Imprimante à impact sérielle selon la revendication 2, dans laquelle lesdites extrémités dudit arbre (60) supportent des roulements (62, 64) et lesdits roulements sont maintenus contre lesdits sièges (84) en forme de cale par ledit moyen élastique (86, 88).
4. Imprimante à impact sérielle selon l'une quelconque des revendications 1 à 3, dans laquelle ledit moyen élastique comprend un rouleau

- (86) sollicité par ressort à chaque extrémité dudit arbre (60), chacun desdits rouleaux étant mobile entre une première position pour capturer une extrémité dudit arbre et une seconde position pour libérer ladite extrémité dudit arbre.
5. Imprimante à impact sérielle selon la revendication 4, dans laquelle chacun desdits rouleaux (86) est sollicité directement contre lesdites extrémités dudit arbre (60) de manière à éliminer tout jeu entre ledit arbre (60) et lesdits roulements (62, 64).
10. 6. Imprimante à impact sérielle selon la revendication 5, dans laquelle chacun desdits rouleaux est sollicité directement contre lesdites extrémités dudit arbre, à l'extérieur desdits roulements.
15. 7. Imprimante à impact sérielle selon l'une quelconque des revendications 4 à 6, dans laquelle ledit moyen élastique comprend un bras (88) pouvant pivoter sur lequel ledit rouleau (86) est monté, un ressort (92) relié audit bras pour solliciter ce bras dans la direction de capture, et un bras (98) de loquet pour déplacer ledit bras mobile dans la direction de libération, opposée à ladite direction de capture.
20. 8. Imprimante à impact sérielle selon la revendication 7, dans laquelle ledit bras (98) de loquet peut pivoter et supporter une came d'entraînement (96) sur son dessus pour une action mutuelle avec un galet de came (94) sur ledit bras pouvant pivoter (88) pour déplacer ce bras dans la direction de libération.
25. 9. Imprimante à impact sérielle comprenant une platine (10) pour supporter un récepteur d'image, ladite platine comportant un bras de support central (60) et un manchon cylindrique absorbant les chocs (54, 56) qui lui est fixé, un élément d'impression (48) ayant des parties d'impression de caractères disposées sur son dessus, un sélecteur d'élément d'impression (46) pour déplacer ledit élément d'impression de manière à positionner une partie sélectionnée de caractère au droit d'une position d'impression, un ruban de marquage pouvant être placé entre ledit élément d'impression et ladite platine, et un moyen (30) pour fournir une force à ladite partie de caractère de manière à l'entraîner contre ladite platine, ledit moyen de fourniture de force ayant une masse effective d'au moins 0,23 kg et étant agencé de manière à déformer ladite platine pendant une durée de contact d'au moins 1 milliseconde, et où ladite
30. 40. 45. 50. 55.

imprimante est caractérisée par :

un moyen pour capturer les extrémités du-
dit arbre (60), ledit moyen de capture compor-
tant des blocs de support (82) s'étendant à
partir de la base de ladite imprimante à cha-
que extrémité dudit arbre, chaque bloc définis-
sant un siège en forme de cale (84), lesdits
blocs étant capables de supporter, sans défor-
mation, une force d'impression par impact en
provenance dudit moyen de fourniture de for-
ce, délivrée dans une première direction, et un
moyen (86, 88) pour retenir les extrémités du-
dit arbre (60) en contact avec lesdits blocs de
support (87) avec une force suffisante pour
vaincre le déplacement des extrémités dudit
arbre dans une seconde direction sensiblement
opposée à ladite première direction,

des roulements (62, 64) supportés sur les
extrémités dudit arbre pour assise dans lesdits
sièges en forme de cale, et

un moyen (76, 80) pour solliciter axiale-
ment ledit arbre de manière à éliminer le jeu
axial entre ledit arbre et lesdits roulements et
éliminer le glissement axial de la platine en
fournissant une force d'entraînement de ma-
nière à solliciter ledit arbre dans une direction
axiale.

5

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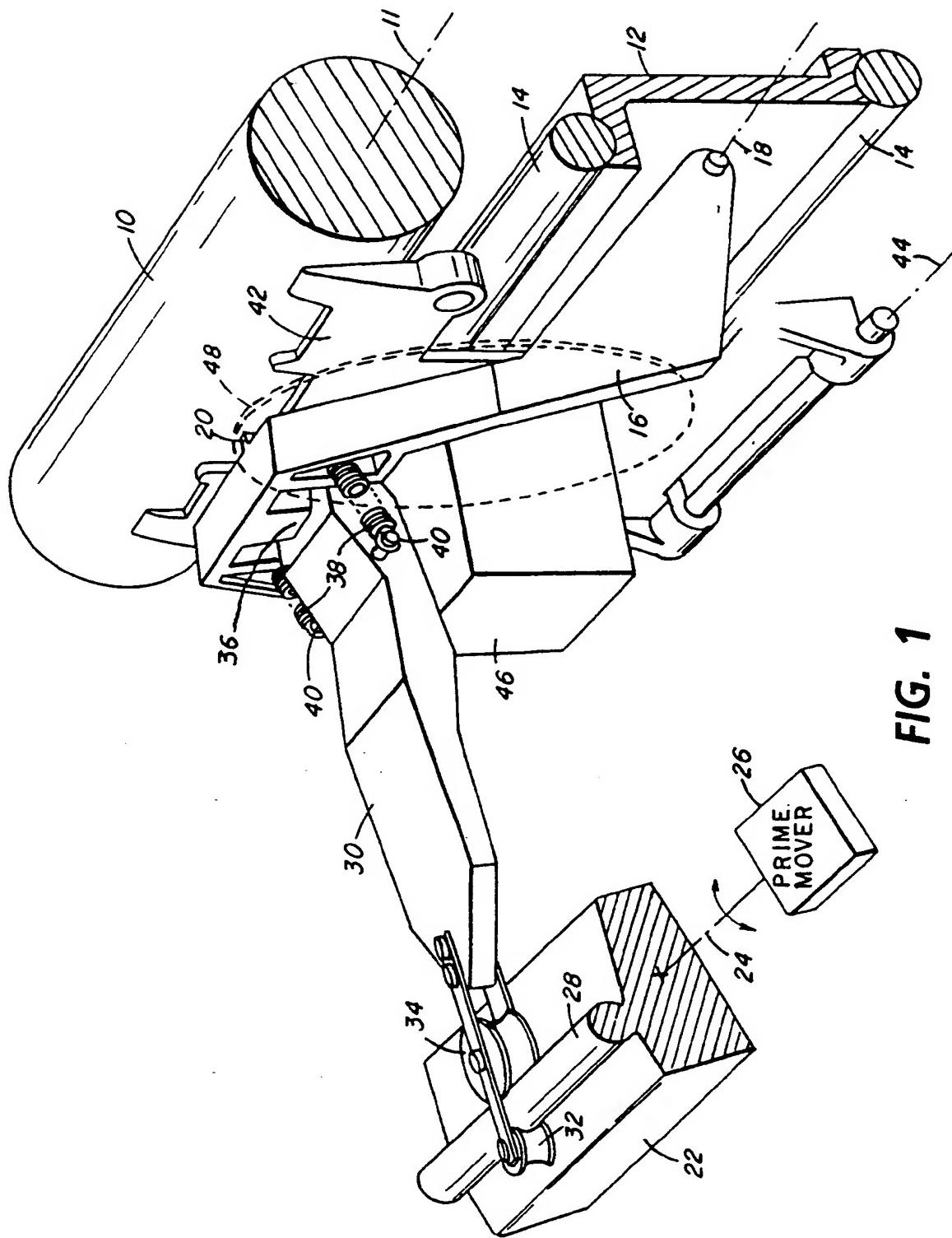
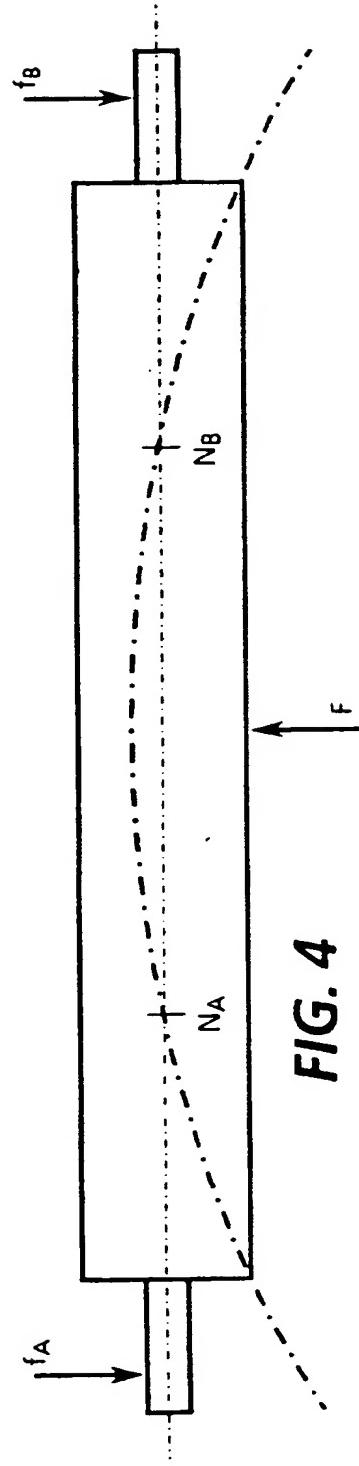
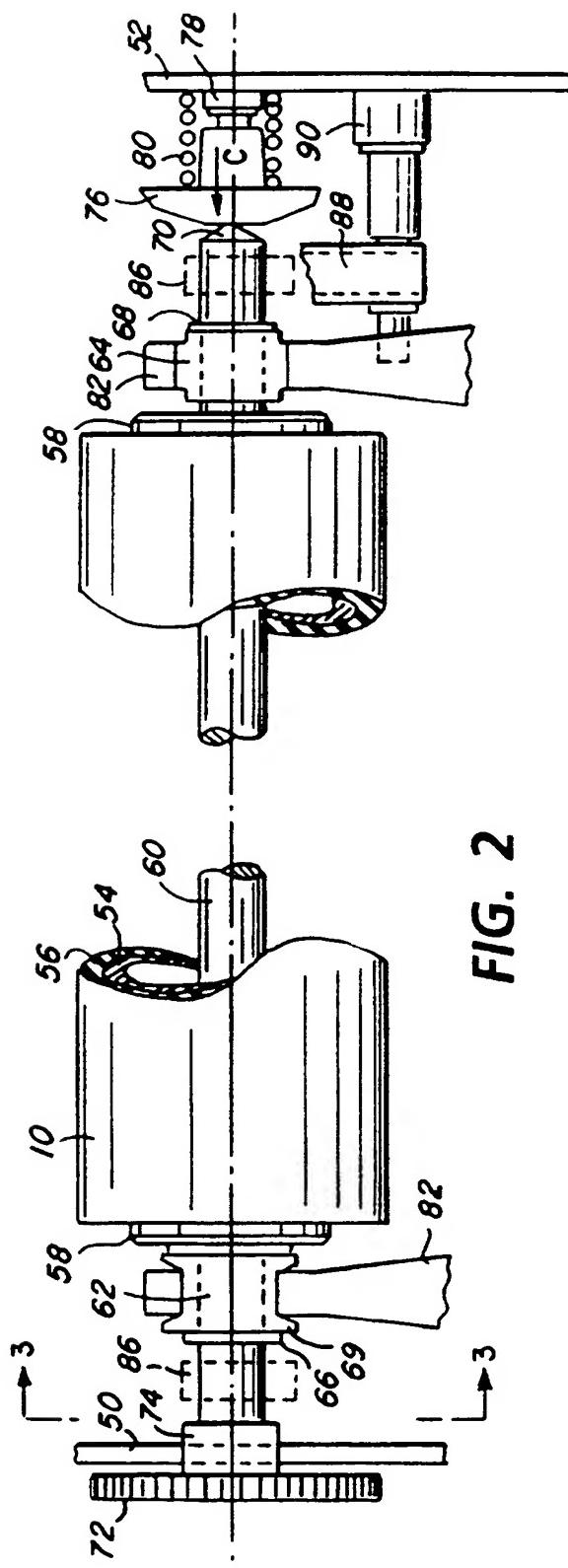


FIG. 1



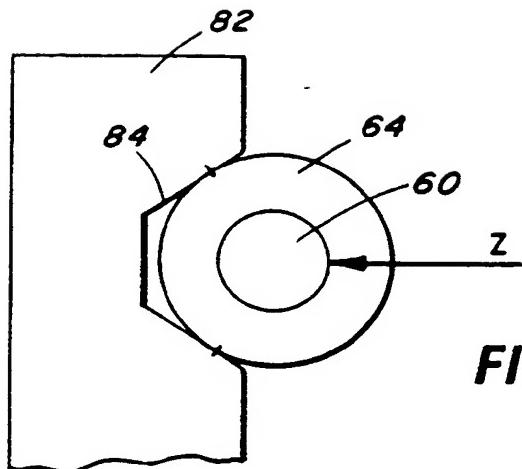


FIG. 3A

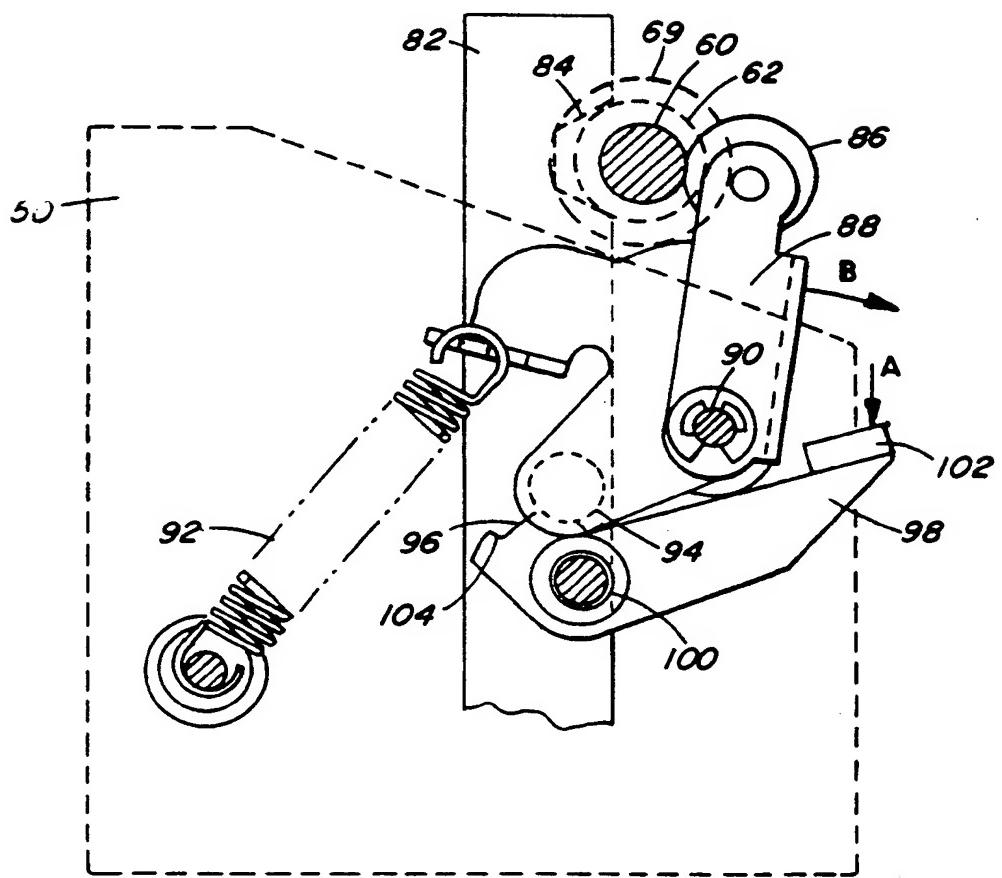


FIG. 3